มหาวิทยาลัยสงขลานครินทร์ คณะวิศวกรรมศาสตร์

สอบปลายภาค ประจำภาคการศึกษา 1	ปีการศึกษา 254
วันที่ 10/10/06	เวลา 9.00-12.00
วิชา 220-501 Matrix Structural Analysis	
ห้องสอบ A203	
ชื่อ-สกุล	รหัส

คำชี้แจง

- 1.ข้อสอบทั้งหมดมี 6 ข้อ คะแนนรวม 120 คะแนน ดังแสดงในตารางข้างล่าง
- 2.ข้อสอบมีทั้งหมด 4 หน้า (ไม่รวมปก) ผู้สอบด้องตรวจสอบว่ามีครบทุกหน้าหรือไม่ (ก่อนลงมือทำ) และห้ามแกะหรือฉีกข้อสอบออกจากเล่ม
- 3.ให้ทำหมคทุกข้อลงในสมุคคำตอบ
- 4.อนุญาตให้ใช้เครื่องคิดเลขได้ทุกชนิด
- 5.กระดาษทดที่แจกให้ไม่ต้องส่งคืน ถ้าไม่พอขอเพิ่มที่อาจารย์คุมสอบ
- 6.ห้ามหยิบ หรือยืมสิ่งของใดๆ ของผู้อื่นในห้องสอบ
- 7. One Page of Note Allowed

8. GOOD LUCK

ตารางคะแนน

ข้อที่	กะแนนเต็ม	ได้
1	20	
2	20	
3	20	
4	20	
5	20	
6	20	
รวม	120	

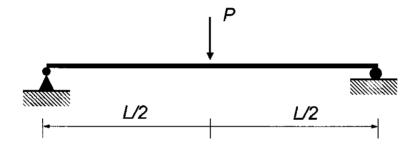
Lecturer: Asst. Prof. Dr. Suchart Limkatanyu

Problem 1. (20 points)

For the uniform simply supported beam carrying a central point load as shown below, use the **Principle of Virtual Displacements** with the given assumed displacement as:

$$v(x) = a_1 \sin\left(\frac{\pi x}{L}\right) + a_2 \sin\left(\frac{3\pi x}{L}\right)$$

- (1) Verify that this given displacement field is acceptable.
- (2) Find the vertical displacement at mid-span (It should be larger or smaller that the exact value, give the supporting reason).
- (3) Find the bending moment at the mid-span section.
- (4) Find the reaction forces at the left and right ends.



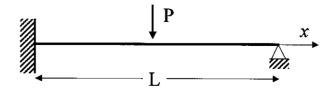
Note:

$$\int_{0}^{L} \sin\left(\frac{m\pi x}{L}\right) \sin\left(\frac{n\pi x}{L}\right) dx = \frac{L}{2} : if \ m = n$$

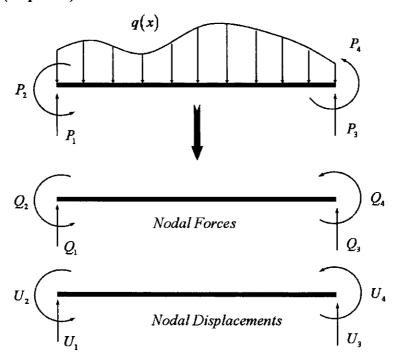
$$= 0 : if \ m \neq n$$

Problem 2. (20 points)

- (1) In your own words, explain the difference between strong form and weak form of a beam problem.
- (2) In your own words, explain the difference between the Rayleigh-Ritz method and the displacement-based finite element method.
- (3) Consider the beam shown below. Consider a Rayleigh-Ritz solution of the form $v(x) = c_1 \phi(x)$. What are the conditions that $\phi(x)$ must satisfy? Suggest an acceptable expression for $\phi(x)$.



Problem 3. (20 points)



For the beam subjected to the distributed load q(x) and the end forces P, show equivalence between principle of Virtual Displacements, and Principle of Stationary Potential Energy. In other words, show that both principles lead to the same equilibrium equation:

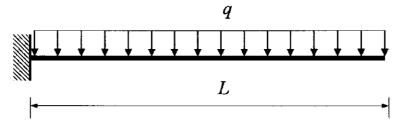
$$\mathbf{K}_{ele}\mathbf{U} = \mathbf{Q}$$

where

$$\mathbf{K}_{ele} = \int_{L} \mathbf{B}^{T}(x) IE(x) \mathbf{B}(x) dx$$

$$\mathbf{Q} = \mathbf{P} + \int_{L} \mathbf{N}^{T} q(x) dx$$

Problem 4: (20 points)



Given assumed displacement as:

$$v(x) = c_2 x^2 + c_3 x^3 + c_4 x^4$$

For the uniform cantilever beam carrying a uniformly distributed load as shown above, use the **Principle of Minimum Potential Energy** to show that the generalized stiffness equations are given as:

$$\frac{IE}{2} \begin{bmatrix} 8L & 12L^2 & 16L^3 \\ 12L^2 & 24L^3 & 36L^4 \\ 16L^3 & 36L^4 & \frac{288}{5}L^5 \end{bmatrix} \begin{bmatrix} c_2 \\ c_3 \\ c_4 \end{bmatrix} = \begin{cases} \frac{qL^3}{3} \\ \frac{qL^4}{4} \\ \frac{qL^5}{5} \end{bmatrix}$$

Will this assumed displacement result in the exact solution?

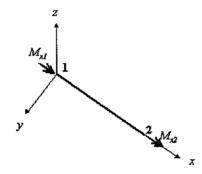
What will happen if you include the term c_5x^5 into the assumed displacement field?

Problem 5: (20 points)

Consider the two-node element shown below. The torsional response of the element is of interest. The nodal displacements are $\mathbf{U} = \{\theta_{x1} \quad \theta_{x2}\}^T$, the corresponding nodal forces are $\mathbf{P} = \{M_{x1} \quad M_{x2}\}^T$. The cross section is circular and the radius changes according to $R(x) = R_0 (1 - x/2L)$, where R_0 is the radius at x=0.

Consider a displacement-based formulation. Write down the expression for the element stiffness matrix, and specify the expressions for each term that appears in the stiffness matrix. Do not solve the integrals!!!

Do you expect any of the above formulations to be "exact"? Why?



Problem 6: (20 points)

Consider the 2-node beam element shown. It has 3 degrees of freedom per node: vertical displacement v_i , rotation θ_i and curvature κ_i (i=1,2). The vertical displacement field v(x) is written in terms of the shape function $N_i(x)$ and the nodal displacements \mathbf{d} as: $v(x) = \mathbf{N}(x) \mathbf{d} = \begin{bmatrix} N_1(x) & N_2(x) & N_3(x) & N_4(x) & N_5(x) & N_6(x) \end{bmatrix} \begin{bmatrix} v_1 & \theta_1 & \kappa_1 & v_2 & \theta_2 & \kappa_2 \end{bmatrix}^T$

- a) What is the order of the six polynomials $N_i(x)$, i=1,6?
- b) Show the procedure you would follow to determine the interpolation functions (do not derive them!!)
- c) What is the main drawback of the curvature continuity of this element?

